

RAIRO. ANALYSE NUMÉRIQUE

CLAES JOHNSON

An elasto-plastic contact problem

RAIRO. Analyse numérique, tome 12, n° 1 (1978), p. 59-74

http://www.numdam.org/item?id=M2AN_1978__12_1_59_0

© AFCET, 1978, tous droits réservés.

L'accès aux archives de la revue « RAIRO. Analyse numérique » implique l'accord avec les conditions générales d'utilisation (<http://www.numdam.org/conditions>). Toute utilisation commerciale ou impression systématique est constitutive d'une infraction pénale. Toute copie ou impression de ce fichier doit contenir la présente mention de copyright.

NUMDAM

Article numérisé dans le cadre du programme
Numérisation de documents anciens mathématiques
<http://www.numdam.org/>

AN ELASTO-PLASTIC CONTACT PROBLEM (*) (1)

by Claes JOHNSON (2)

Communiqué par P. G. Ciarlet

Abstract. — We study the problem of finding the stresses and the displacements in an elasto-plastic body \mathcal{E} in frictionless contact with a rigid body which is pressed against \mathcal{E} . We prove existence of a solution and then we consider finite element methods for finding approximate solutions of the problem.

INTRODUCTION

Duvaut [1] has studied the problem of finding the stresses in an elasto-plastic body \mathcal{E} in frictionless contact with a rigid body \mathcal{B} which is pressed against \mathcal{E} . In this note we extend the study of Duvaut by looking also for the displacements of \mathcal{E} and \mathcal{B} . We shall consider a stationary case corresponding to Henky's law. For simplicity we shall assume that \mathcal{E} is isotropic.

In Section 1 we prove existence of a solution to the contact problem assuming that \mathcal{E} is elastic-perfectly plastic. In this case the displacements of \mathcal{E} may be discontinuous (and even non-unique) and we have to use a formulation requiring little regularity of the displacements. One way of obtaining more regular displacements is to assume a suitable hardening of the elasto-plastic material. Such a case is studied in Section 2. Then in Sections 3 and 4 we consider finite element methods for finding approximate solutions of the contact problem.

1. ELASTIC-PERFECTLY PLASTIC MATERIAL--

Suppose that initially the elasto-plastic body \mathcal{E} occupies the bounded region $\Omega \subset \mathbf{R}^3$ with boundary Γ and that Γ contains an open set Γ_1 in the plane $\{x = (x_1, x_2, x_3) \in \mathbf{R}^3 : x_3 = 0\}$. Moreover, suppose that initially the rigid body \mathcal{B} occupies the region

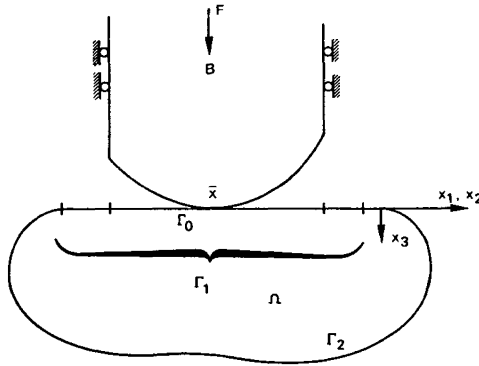
$$B = \{x \in \mathbf{R}^3 : -x_3 \geq \varphi(x_1, x_2), (x_1, x_2) \in \bar{\Gamma}_0\},$$

where Γ_0 is an open set compactly contained in Γ_1 with smooth boundary and $\varphi : \Gamma_1 \rightarrow \mathbf{R}$ is smooth, nonnegative and $\varphi(\bar{x}) = 0$ for some $\bar{x} \in \Gamma_0$ (see *Fig.*).

(*) Reçu avril 1977, révisé août 1977.

(1) Texte d'une conférence présentée aux Journées « Éléments Finis », Rennes, 4-6 mai 1977

(2) Chalmers University of Technology, Department of Computer Sciences, Göteborg, Sweden.



Let the boundary of \mathcal{E} be fixed on the portion $\Gamma_2 = \Gamma \setminus \Gamma_1$ and free on Γ_1 . Let \mathcal{B} be acted upon by the vertical force $F (F > 0)$ and suppose that \mathcal{B} is free to move vertically, whereas rotation and horizontal displacement are prevented. We want to find the vertical displacement U of \mathcal{B} , the stress $\sigma = \{ \sigma_{ij} \}$, $i, j = 1, 2, 3$, in \mathcal{E} , and the displacement $u = \{ u_i \}$, $i = 1, 2, 3$, of \mathcal{E} , where u_i is the displacement in the x_i -direction. The reference configuration is the one in *figure*. We shall assume that the displacements are small; in particular this means that the relation $u_3 \cong U - \varphi$ can be used to describe the compatibility of the displacements of \mathcal{B} and \mathcal{E} .

We shall use the following notation: For m a positive integer and $1 \leq p \leq \infty$, let $\| \cdot \|_{m,p}$ denote the norm in the usual Sobolev space $[W_p^m(\Omega)]^n$ with n a positive integer. If $m = 0$ we omit this index and write $\| \cdot \|_p$ instead of $\| \cdot \|_{0,p}$. Let (\cdot, \cdot) and $\| \cdot \|$ denote the scalar product and norm in $[L^2(\Omega)]^n$. Further we define

$$H = \{ \tau = \{ \tau_{ij} \} \in [L^2(\Omega)]^9 : \tau_{ij} = \tau_{ji}, i, j = 1, 2, 3, \},$$

$$\mathcal{W} = [W]^3, \quad W = \{ w \in W_2^1(\Omega) : w = 0 \text{ on } \Gamma_2 \},$$

$$K = \{ (u, U) \in \mathcal{W} \times \mathbf{R} : u_3 + \varphi - U \geq 0 \text{ on } \bar{\Gamma}_0 \},$$

and the deformation $\varepsilon(u) = \{ \varepsilon_{ij}(u) \}$ associated with $u \in \mathcal{W}$ by

$$\varepsilon_{ij}(u) = \frac{1}{2}(u_{i,j} + u_{j,i}), \quad i, j = 1, 2, 3,$$

where $w_{,j} = \partial w / \partial x_j$. We recall Green's formula:

$$(\tau, \varepsilon(v)) = \int_{\Gamma} \tau_{ij} n_j u_i ds - (\text{div } \tau, v), \tag{1.1}$$

where $n = (n_1, n_2, n_3)$ is the outward unit normal to Γ and

$$\begin{aligned} \operatorname{div} \tau &= ((\operatorname{div} \tau)_i), \quad i = 1, 2, 3, \\ (\operatorname{div} \tau)_i &= \tau_{ij,j}. \end{aligned}$$

Here and below we use the summation convention: repeated indices indicate summation from 1 to 3.

Let $D \subset \mathbf{R}^9$ be a given closed convex set with $0 \in D$ and define the set of plastically admissible stresses

$$P = \{ \tau \in H : \tau^d(x) \in D \text{ a.e. in } \Omega \},$$

where

$$\begin{aligned} \tau^d &= \tau - \frac{1}{3} \operatorname{tr}(\tau) \delta, \\ \operatorname{tr}(\tau) &= \tau_{kk}, \\ \delta &= \{ \delta_{ij} \}, \quad \delta_{ij} = \begin{cases} 1 & \text{if } i = j, \\ 0 & \text{if } i \neq j. \end{cases} \end{aligned}$$

The tensor τ^d is the so called *stress deviatoric* associated to τ . If $\sigma(x) \in$ (interior of D), then \mathcal{E} is in an elastic state at the point $x \in \Omega$ and then we have the linear strain-stress relation

$$\begin{aligned} \varepsilon(u) &= A \sigma, \\ (A \sigma)_{ij} &= \lambda \operatorname{tr}(\sigma) \delta_{ij} + \nu \sigma_{ij}^d, \end{aligned}$$

where λ and ν are positive constants. For notational simplicity we shall assume below that $\nu = 1$. We define the bilinear form

$$a(\sigma, \tau) = \int_{\Omega} (\lambda \operatorname{tr}(\sigma) \operatorname{tr}(\tau) + \sigma_{ij}^d \tau_{ij}^d) dx,$$

and we note that

$$a(\tau, \tau) \geq \alpha \|\tau\|^2, \quad \tau \in H, \tag{1.2}$$

where $\alpha = \min(1, 9\lambda)$.

A natural formulation of the contact problem is now the following: Find $(\sigma, (u, V)) \in P \times K$ such that

$$a(\sigma, \tau - \sigma) - (\varepsilon(u), \tau - \sigma) \geq 0, \quad \forall \tau \in P, \tag{1.3 a}$$

$$(\sigma, \varepsilon(v - u)) \geq F(V - U), \quad \forall (v, V) \in K, \tag{1.3 b}$$

or, equivalently, find a saddle point $(\sigma, (u, V)) \in P \times K$ for the functional $L : P \times K \rightarrow \mathbf{R}$ defined by

$$L(\tau, (v, V)) = \frac{1}{2} \|\tau\|_a^2 - (\tau, \varepsilon(v)) + FV,$$

where $\| \cdot \|_a^2 = a(\cdot, \cdot)$. However, the regularity of the displacement u needed in this formulation is in general not possible to achieve in the perfectly-plastic case. Therefore, we shall instead consider a formulation requiring less regularity of u (cf. [3]). To be more precise, we shall seek u in the space $Y_{3/2}$, where for $1 \leq p \leq \infty$,

$$Y_p = [L_p(\Omega)]^3.$$

To motivate this formulation we first note that by Green's formula (1.1), it follows that (1.3 b) is formally equivalent to the following relations:

$$\operatorname{div} \sigma = 0 \quad \text{in } \Omega, \quad (1.4)$$

$$-\int_{\Gamma_0} \sigma_{33} ds = F, \quad (1.5)$$

$$\sigma_{13} = \sigma_{23} = 0, \quad \sigma_{33} \leq 0 \quad \text{on } \Gamma_1, \quad (1.6)$$

$$\sigma_{33} = 0 \quad \text{on } \Gamma_1 \setminus \Gamma_0, \quad (1.7)$$

$$\sigma_{33}(x) = 0 \quad \text{if } (u_3 + \varphi - U)(x) > 0, \quad x \in \Gamma_0, \quad (1.8)$$

which is the intuitive way of formulating the statical relationship in the contact problem.

We shall seek the stress σ in the space $\mathcal{P}_3 = P \cap \mathcal{H}_3$, where for $2 \leq q < \infty$,

$$\mathcal{H}_q = \{ \tau \in H : \operatorname{div} \tau \in Y_q \text{ and } \tau \text{ satisfies (1.6) and (1.7)} \}.$$

Here (1.6) and (1.7) are to be understood in the following sense:

$$\int_{\Gamma} \tau_{13} w ds = \int_{\Gamma} \tau_{23} w ds = 0, \quad w \in \mathcal{W}, \quad (1.9)$$

$$\int_{\Gamma} \tau_{33} w ds \leq 0, \quad w \in \mathcal{W}, \quad w \geq 0, \quad (1.10)$$

$$\int_{\Gamma} \tau_{33} w ds = 0 \quad \text{for } w \in \mathcal{W} \text{ such that } w = 0 \text{ on } \Gamma_0. \quad (1.11)$$

Note that if $\tau \in H$ and $\operatorname{div} \tau \in Y_2$, then (see [2]) $\tau_{ij} n_j \in H^{-\frac{1}{2}}(\Gamma)$ so that (1.9)-(1.11) are meaningful. Now, taking $\tau \in \mathcal{P}_3$ in (1.3 a) using Green's formula and (1.6), we find with $\psi \in \mathcal{W}$ satisfying $\psi = 1$ on Γ_0 , that

$$\begin{aligned} 0 &\leq a(\sigma, \tau - \sigma) - (\varepsilon(u), \tau - \sigma) \\ &= a(\sigma, \tau - \sigma) + (u, \operatorname{div} \tau - \operatorname{div} \sigma) + \int_{\Gamma} u_3 (\tau_{33} - \sigma_{33}) ds \\ &= a(\sigma, \tau - \sigma) + (u, \operatorname{div} \tau - \operatorname{div} \sigma) + \int_{\Gamma} (u_3 + \psi(\varphi - U)) (\tau_{33} - \sigma_{33}) ds \\ &\quad + \int_{\Gamma} \psi (U - \varphi) (\tau_{33} - \sigma_{33}) ds, \end{aligned}$$

so that

$$a(\sigma, \tau - \sigma) + (u, \operatorname{div} \tau - \operatorname{div} \sigma) + \int_{\Gamma} \psi(U - \varphi)(\tau_{33} - \sigma_{33}) ds \geq 0, \quad (1.12)$$

since by (1.7)-(1.8),

$$\int_{\Gamma} (u_3 + \psi(\varphi - U)) \sigma_{33} ds = 0,$$

and by (1.10) and (1.11) assuming that $(u, U) \in K$,

$$\int_{\Gamma} (u_3 + \psi(\varphi - U)) \tau_{33} ds \leq 0.$$

We are thus led to the following formulation of the contact problem: Find $(\sigma, u, U) \in \mathcal{P} \times Y \times \mathbf{R}$ such that

$$a(\sigma, \tau - \sigma) + (u, \operatorname{div} \tau - \operatorname{div} \sigma) + \int_{\Gamma} \psi(U - \varphi)(\tau_{33} - \sigma_{33}) ds \geq 0, \quad \tau \in \mathcal{P}, \quad (1.13 a)$$

$$(v, \operatorname{div} \sigma) = 0, \quad v \in Y, \quad (1.13 b)$$

$$-\int_{\Gamma} \psi \sigma_{33} ds = F, \quad (1.13 c)$$

where $\mathcal{P} = \mathcal{P}_3$ and $Y = Y_{3/2}$.

Remark: Note that the condition $u_3 + \varphi - U \geq 0$ on Γ_0 does not appear explicitly in this formulation, which is natural since the trace of $u \in Y$ on Γ may not be defined.

To prove existence of a solution of (1.13) we shall need the following "safe load hypothesis":

$$\begin{aligned} &\text{There exists } \delta > 0 \quad \text{and} \quad \chi \in \mathcal{P} \cap E \\ &\text{such that } \operatorname{dist}(\chi(x), \partial D) > \delta \text{ for } x \in \Omega, \end{aligned} \quad (1.14)$$

where ∂D denotes the boundary of D and

$$E = \left\{ \tau \in H : \operatorname{div} \tau = 0 \text{ in } \Omega, -\int_{\Gamma} \psi \tau_{33} = F \right\}.$$

Note that with $\delta = 0$, this is a necessary condition for existence of a solution.

THEOREM 1: *If (1.14) holds then there exists $(\sigma, u, U) \in \mathcal{P} \times Y \times \mathbf{R}$, satisfying (1.13). Moreover σ is uniquely determined.*

Proof: The proof will be divided into three parts: First we prove existence of a solution of a regularized problem depending on a parameter $\mu > 0$. Then we establish some *a priori* estimates for the solution of this problem and finally we obtain a solution of the original problem by passing to the limit as μ tends to zero. The uniqueness of σ is easy to prove.

(a) The regularized problem

For $\mu > 0$ we consider the following problem: Find a saddle point $(\sigma_\mu, (u_\mu, U_\mu)) \in H \times K$ for the regularized Lagrangian $L_\mu : H \times K \rightarrow \mathbf{R}$ defined by

$$L_\mu(\tau, (v, V)) = \frac{1}{2} \|\tau\|_a^2 + J_\mu(\tau) - (\tau, \varepsilon(v)) + FV,$$

where

$$J_\mu(\tau) = \frac{1}{2\mu} \|\tau - \pi\tau\|^2,$$

and π is the orthogonal projection in \mathbf{R}^9 onto D . In other words, we seek $(\sigma_\mu, (u_\mu, U_\mu)) \in H \times K$ satisfying

$$a(\sigma_\mu, \tau) + (J'_\mu(\sigma_\mu), \tau) - (\tau, \varepsilon(u_\mu)) = 0, \quad \tau \in H, \quad (1.15a)$$

$$(\sigma_\mu, \varepsilon(v - u_\mu)) \geq F(V - U_\mu), \quad (v, V) \in K, \quad (1.15b)$$

where $J'(\tau) = (1/\mu)(\tau - \pi\tau)$. Existence of a saddle point $(\sigma_\mu, (u_\mu, U_\mu))$ will follow easily if we can show that the problem

$$\sup_{(v, V) \in K} g_\mu(v, V), \quad (1.16)$$

where

$$g_\mu(v, V) = \inf_{\tau \in H} L_\mu(\tau, (v, V)), \quad (1.17)$$

has a solution. The infimum in (1.17) is attained for $\tau = \bar{\tau}$ satisfying

$$\varepsilon(v) - A\bar{\tau} = \frac{1}{\mu}(\bar{\tau} - \pi\bar{\tau}),$$

i. e.,

$$\varepsilon(v)^d - \bar{\tau}^d = \frac{1}{\mu}(\bar{\tau}^d - \pi\bar{\tau}^d), \quad (1.18)$$

$$\text{tr}(\varepsilon(v)) - \lambda \text{tr}(\bar{\tau}) = 0,$$

since by the definition of D , $\text{tr}(\tau - \pi\tau) = 0$ for $\tau \in H$. But (1.18) implies that $\pi\bar{\tau}^d = \pi\varepsilon(v)^d$ and thus

$$\bar{\tau} = \frac{1}{\lambda} \text{tr}(\varepsilon(v)) + \frac{\mu}{1+\mu} \varepsilon(v)^d + \frac{1}{1+\mu} \pi\varepsilon(v)^d,$$

which gives after a simple computation

$$g_\mu(v, V) = \frac{1}{2(1+\mu)} \|\varepsilon(v)^d - \pi\varepsilon(v)^d\|^2 - \frac{1}{2} \|\varepsilon(v)^d\|^2 - \frac{1}{2\lambda} \|\text{tr}(\varepsilon(v))\|^2 + FV.$$

Since $g_\mu : \mathcal{W} \times \mathbf{R} \rightarrow \mathbf{R}$ is concave (being the infimum of a set of linear functions) and continuous and K is closed and convex in $\mathcal{W} \times \mathbf{R}$, to prove existence of a solution of the problem (1.16) it remains only to prove that g_μ is coercive, i. e.,

$$g_\mu(v, V) \rightarrow -\infty \quad \text{as} \quad \|(v, V)\|_{\mathcal{W} \times K} \rightarrow \infty, \quad (v, V) \in \mathcal{W} \times K.$$

But this follows easily from Korn's inequality (see [2]),

$$\|v\|_{\mathcal{W}} \leq C \|\varepsilon(v)\|, \quad v \in \mathcal{W},$$

the trace inequality,

$$\|v_3\|_{L^1(\Gamma_0)} \leq C \|v\|_{\mathcal{W}}, \quad v \in \mathcal{W},$$

and the fact that

$$V \leq v_3 + \varphi \quad \text{on} \quad \Gamma_0,$$

if $(v, V) \in K$. Thus the problem (1.16) has a solution $(u_\mu, U_\mu) \in K$.

The extremality relation can be written

$$(\sigma_\mu, \varepsilon(v - u_\mu)) \geq F(V - U_\mu), \quad (v, V) \in K,$$

with

$$\sigma_\mu = \frac{1}{\lambda} \text{tr}(\varepsilon(u_\mu)) + \frac{\mu}{1+\mu} \varepsilon(u_\mu)^d + \frac{1}{1+\mu} \pi\varepsilon(u_\mu)^d.$$

Thus (1.15 b) holds and it is easy to check that also (1.15 a) is satisfied and therefore $(\sigma_\mu, (u_\mu, U_\mu)) \in H \times K$ is a saddle point for L_μ .

(b) A priori estimates

By varying $(v, V) \in X$ in (1.15 b) one concludes that σ_μ satisfies the relations (1.9)-(1.11) and

$$\text{div} \sigma_\mu = 0 \quad \text{in} \quad \Omega, \tag{1.19}$$

$$-\int_{\Gamma_1} \psi \sigma_{\mu, 33} ds = 0, \tag{1.20}$$

$$\int_{\Gamma_1} (u_{\mu, 3} + \psi(\varphi - U_\mu)) \sigma_{\mu, 33} ds = 0. \tag{1.21}$$

Thus, replacing τ in (1.15 a) by $\tau - \sigma_\mu$ where $\tau \in \mathcal{P}$ and applying Green's formula, paralleling the proof of (1.12) we find that

$$a(\sigma_\mu, \tau - \sigma_\mu) + (J'_\mu(\sigma_\mu), \tau - \sigma_\mu) + (u_\mu, \operatorname{div} \tau - \operatorname{div} \sigma_\mu) + \int_\Gamma \psi(U_\mu - \varphi)(\tau_{33} - \sigma_{33, \mu}) ds \geq 0. \quad (1.22)$$

If we now take $\tau = \chi$, where χ is given by assumption (1.14) and use the fact σ_μ as well as χ satisfies (1.19) and (1.20), we get

$$\begin{aligned} & \|\sigma_\mu\|_a^2 + (J'_\mu(\sigma_\mu), \sigma_\mu - \chi) \\ & \leq \int_\Gamma \psi \varphi (\chi_{33} - \sigma_{33, \mu}) ds + a(\sigma_\mu, \chi). \end{aligned} \quad (1.23)$$

But, as is easily seen, (1.14) implies that

$$\|J'_\mu(\sigma_\mu)\|_1 \leq \frac{1}{\delta} (J'_\mu(\sigma_\mu), \sigma_\mu - \chi).$$

Using also the estimate (see [2]):

$$\|\tau\|_{H^{-\frac{1}{2}}(\Gamma)} \leq C(\|\tau\| + \|\operatorname{div} \tau\|), \quad (1.24)$$

and (1.19), we now conclude from (1.23) that

$$\|\sigma_\mu\| \leq C(\|\psi \varphi\|_{H^{1/2}(\Gamma)} + \|\chi\|) \leq C, \quad (1.25)$$

$$\|J'_\mu(\sigma_\mu)\|_1 \leq C, \quad (1.26)$$

with C independent of μ . Using the equation (1.15 a), it then follows that

$$\|\varepsilon(u_\mu)\|_1 \leq C, \quad (1.27)$$

so that

$$\|u_\mu\|_{3/2} \leq C, \quad (1.28)$$

by using the estimate

$$\|v\|_{3/2} \leq C\|\varepsilon(v)\|_1, \quad v \in \mathcal{W}.$$

A proof of this result in the case $\Gamma_2 = \psi$ can be found in [3]. The proof can easily be modified to cover also the present case.

Further, to bound U_μ we note that by the easy to prove trace inequality

$$\int_{\Gamma_0} |w| ds \leq C\|w, 3\|_1, \quad w \in \mathcal{W},$$

and (1.27), we have

$$\int_{\Gamma_0} |u_{\mu 3}| ds \leq C.$$

Since $U_\mu \leq u_{\mu 3} + \varphi$ on Γ_0 we thus find that $U_\mu \leq C$. Moreover, taking $\tau = \sigma_\mu$ in (1.15 a) and $(v, V) = 0$ in (1.15 b) and adding we see that

$$\|\sigma_\mu\|_a^2 + (J'_\mu(\sigma_\mu), \sigma_\mu) \leq F U_\mu.$$

But since J'_μ is monotone and $J'_\mu(0) = 0$, $(J'_\mu(\sigma_\mu), \sigma_\mu) \geq 0$ and thus $U_\mu \geq 0$ so that

$$|U_\mu| \leq C. \tag{1.29}$$

(c) Passage to the limit

From (1.19), (1.24)-(1.27), (1.28) and (1.29) it follows that there exists $(\sigma, (u, U)) \in \mathcal{P} \times Y \times \mathbf{R}$ with $\text{div } \sigma = 0$ and a sequence μ tending to zero such that

$$\left. \begin{aligned} \sigma_\mu &\rightarrow \sigma \text{ weakly in } H, \\ \sigma_{33, \mu} &\rightarrow \sigma_{33} \text{ weakly in } H^{-1/2}(\Gamma), \\ u_\mu &\rightarrow u \text{ weakly in } Y, \\ U_\mu &\rightarrow U. \end{aligned} \right\} \tag{1.30}$$

Passing to the limit in the relation

$$\begin{aligned} a(\sigma_\mu, \tau - \sigma_\mu) + (u_\mu, \text{div } \tau - \text{div } \sigma_\mu) \\ + \int_\Gamma \psi(U_\mu - \varphi)(\tau_{33} - \sigma_{33, \mu}) ds \geq 0, \quad \tau \in \mathcal{P}, \end{aligned}$$

which follows from (1.22) using the monotonicity of J'_μ , we now obtain (1.13 a) without any difficulty recalling that $\text{div } \sigma_\mu = \text{div } \sigma = 0$. Finally, (1.13 c) follows from (1.30) by passing to the limit in (1.20). This completes the proof.

2. HARDENING MATERIAL

To describe the hardening of the elasto-plastic material (cf. [4]) we shall use a hardening parameter $\xi = \{\xi_i\}$, $i = 1, \dots, m$, where m is a positive integer. We shall use the notation

$$\begin{aligned} \hat{H} &= \{\hat{\sigma} = (\sigma, \xi) : \sigma \in H, \xi \in [L^2(\Omega)]^m\}, \\ [\hat{\sigma}, \hat{\tau}] &= a(\sigma, \tau) + \gamma(\xi, \eta), \quad \hat{\sigma} = (\sigma, \xi), \tau = (\tau, \eta) \in \hat{H}, \\ \|\tau\|_a &= [\hat{\tau}, \hat{\tau}]^{1/2}, \quad \hat{\tau} \in \hat{H}, \end{aligned}$$

where γ is a positive constant. Let now \hat{D} be a closed convex set in R^{9+m} , the set of admissible combinations of stress deviatorics and hardening, such that $0 \in \hat{D}$ and define

$$\hat{P} = \{\hat{\tau} \in \hat{H} : (\tau^d, \eta)(x) \in \hat{D} \text{ a. e. in } \Omega\}.$$

The elasto-plastic contact problem can now be formulated in the following way: Find $(\hat{\sigma}, (u, U)) \in \hat{P} \times X$ such that

$$[\hat{\sigma}, \hat{\tau} - \hat{\sigma}] - (\varepsilon(u), \tau - \sigma) \geq 0, \quad \forall \hat{\tau} \in \hat{P}, \quad (2.1 a)$$

$$(\sigma, \varepsilon(v-u)) \geq F(V-U), \quad \forall (v, V) \in K, \quad (2.1 b)$$

or, equivalently, find a saddle point $(\hat{\sigma}, (u, U))$ for the functional $\hat{L} : \hat{P} \times K \rightarrow \mathbf{R}$ defined by

$$\hat{L}(\hat{\tau}, (v, V)) = \frac{1}{2} \|\hat{\tau}\|_a^2 - (\tau, \varepsilon(v)) + FV.$$

To prove existence of a solution of (2.1) we shall use the same method as that used in Section 1 for the regularized problem (1.15). Thus, we consider the problem

$$\sup_{(v, V) \in K} g(v, V), \quad (2.2)$$

where

$$\begin{aligned} g(v, V) &\equiv \inf_{\hat{\tau} \in \hat{P}} \hat{L}(\hat{\tau}, (v, V)) \\ &= \frac{1}{2} \|\hat{\varepsilon}(v) - \hat{\pi}\hat{\varepsilon}(v)\|^2 - \frac{1}{2} \|\varepsilon(v)\|^2 + FV, \end{aligned}$$

with

$$\hat{\varepsilon}(v) = (\varepsilon(v), 0) \in \hat{H},$$

and $\hat{\pi}$ being the projection in \hat{H} onto \hat{P} . Since $g(v, V)$ is clearly concave and continuous on $\mathcal{W} \times \mathbf{R}$, existence of a solution of (2.2) will follow if g is coercive on K , i. e., if

$$g(v, V) \rightarrow -\infty \quad \text{if} \quad \|(v, V)\|_{\mathcal{W}} \rightarrow \infty, \quad (v, V) \in K. \quad (2.3)$$

A solution $(u, U) \in K$ of (2.2) is characterized by the variational inequality

$$(\hat{\pi}\hat{\varepsilon}(u), \hat{\varepsilon}(v-u)) \geq F(V-U), \quad (v, V) \in K. \quad (2.4)$$

Thus, having a solution (u, U) of (2.2) we obtain $(\hat{\sigma}, (u, U)) \in \hat{P} \times K$ satisfying (2.1) by setting $\hat{\sigma} = \hat{\pi}\hat{\varepsilon}(u)$. We therefore have the following result:

THEOREM 2: *If (2.3) holds, then there exists $(\hat{\sigma}, U) \in \hat{P} \times X$ satisfying (2.1). Moreover, $\hat{\sigma}$ is uniquely determined.*

Remark. It is easy to verify that (2.3) holds in the following two cases important in applications (for definiteness we use here the von Mises yield criterion, cf. [4]).

(i) *Isotropic hardening:* In this case $m = 1$ and

$$\hat{B} = \{(\tau, \eta) \in \mathbf{R}^9 \times \mathbf{R} : |\tau^d| \leq 1 + \gamma\eta\}.$$

(ii) *Kinematic hardening*: In this case $m = 9$ and

$$B = \{(\tau, \eta) \in \mathbf{R}^9 \times \mathbf{R}^9 : |\tau^d - \eta| \leq 1\}.$$

It is easy to see that also (u, U) is uniquely determined in these two cases.

3. FINITE ELEMENT METHODS: HARDENING MATERIAL

We shall only briefly discuss the case of a hardening material assuming that the coercivity condition (2.3) holds. In this case we simply take a finite dimensional subspace \mathcal{W}_h of \mathcal{W} , define $K_h = \mathcal{W}_h \cap K$ and seek a solution (u_h, U_h) of the problem

$$\sup_{(v, V) \in K_h} g(v, V),$$

or, equivalently, we seek $(u_h, U_h) \in K_h$ satisfying

$$(\hat{\pi}\hat{\varepsilon}(u_h), \hat{\varepsilon}(v - u_h)) \geq F(V - U_h), \quad (v, V) \in K_h. \tag{3.1}$$

Since g is coercive it follows that such a (u_h, U_h) exists.

It is also easy to obtain an estimate for $\|\hat{\pi}\hat{\varepsilon}(u) - \hat{\pi}\hat{\varepsilon}(u_h)\|$ in the following way: Using the fact that since \hat{P} is convex,

$$\|\hat{\pi}\hat{\varepsilon}(u) - \hat{\pi}\hat{\varepsilon}(u_h)\|^2 \leq (\hat{\pi}\hat{\varepsilon}(u) - \hat{\pi}\hat{\varepsilon}(u_h), \hat{\varepsilon}(u) - \hat{\varepsilon}(u_h)),$$

and adding (2.4) and (3.1) with $(v, V) = (u_h, U_h)$ in (2.4), we obtain for all $(v, V) \in K_h$,

$$\|\hat{\pi}\hat{\varepsilon}(u) - \hat{\pi}\hat{\varepsilon}(u_h)\|^2 \leq (\hat{\pi}\hat{\varepsilon}(u_h), v - u) + F(U - V).$$

Having an *a priori* estimate of the form

$$\|\hat{\pi}\hat{\varepsilon}(u_h)\| \leq C, \tag{3.2}$$

this will then give an estimate for $\|\hat{\pi}\hat{\varepsilon}(u) - \hat{\pi}\hat{\varepsilon}(u_h)\|$. In the particular cases discussed in Section 2, (3.2) is easily seen to hold and in these cases we also have

$$\|\varepsilon(u) - \varepsilon(u_h)\| \leq C \|\hat{\pi}\hat{\varepsilon}(u) - \hat{\pi}\hat{\varepsilon}(u_h)\|.$$

4. FINITE ELEMENT METHODS: PERFECTLY PLASTIC MATERIAL

We shall now consider a finite element method based on the formulation (1.13). We shall restrict ourselves to a two-dimensional problem (plane stress or plane strain) and we shall then use the notation of Section 1 with the obvious change from three to two dimensions. The x_2 -axis will now correspond to the x_3 -axis in the three-dimensional case. For simplicity we shall assume that Γ_1 and Γ_0 are line segments (cf. *Fig.*). In the two-dimensional case Theorem 1 holds with $\mathcal{P} = \mathcal{P}_2$ and $Y = Y_2$.

The finite element method will be the following: Given finite dimensional spaces $\mathcal{P}_h \subset \mathcal{P}_2$ and $Y_h \subset Y_2$, find $(\sigma_h, u_h, U_h) \in \mathcal{P}_h \times Y_h \times \mathbf{R}$ such that

$$a(\sigma_h, \tau - \sigma_h) + (u_h, \operatorname{div} \tau - \operatorname{div} \sigma_h) + \int_{\Gamma} \psi(U_h - \varphi)(\tau_{22} - \sigma_{22,h}) ds \geq 0, \quad \tau \in \mathcal{P}_h, \quad (4.1 a)$$

$$(v, \operatorname{div} \sigma_h) = 0, \quad v \in Y_h, \quad (4.1 b)$$

$$-\int_{\Gamma} \psi \sigma_{22,h} ds = 0. \quad (4.1 c)$$

We shall now consider a particular choice of the space \mathcal{P}_h and Y_h . For simplicity we shall assume that Ω is polygonal. Let $\{\mathcal{C}_h\}$, $0 < h < 1$, be a regular family of triangulations of Ω

$$\Omega = \bigcup_{K \in \mathcal{C}_h} K,$$

indexed by the parameter h denoting as usual the maximum of the diameters of the triangles $K \in \mathcal{C}_h$. We assume that nodes are placed at the endpoints of Γ_0 and Γ_1 . We shall construct a finite dimensional space $\mathcal{H}_h \subset \mathcal{H}_2$ and then define $\mathcal{P}_h = \mathcal{H}_h \cap P$. The finite element method will be an equilibrium method, i. e., the spaces \mathcal{H}_h and Y_h will satisfy:

$$\text{If } \tau \in \mathcal{H}_h \text{ and } (\operatorname{div} \tau, v) = 0 \text{ for } v \in Y_h, \text{ then } \operatorname{div} \tau = 0 \text{ in } \Omega. \quad (4.2)$$

Methods of this type including the present one have been studied in [5] in the case of linear elasticity. To define \mathcal{H}_h each triangle K is divided into three subtriangles T_k , $k = 1, 2, 3$, by connecting the center of gravity with the three nodes of K . For each $K \in \mathcal{C}_h$ we introduce the finite dimensional space H_K defined by

$$H_K = \{ \tau = \{ \tau_{ij} \} : \tau_{ij} = \tau_{ji} \text{ is linear on } T_k, \\ k = 1, 2, 3, i, j = 1, 2, \text{ and } \operatorname{div} \tau \in [L^2(K)]^2 \}.$$

One can prove (see [5]) that an element $\tau \in H_K$ is uniquely determined by the following 15 degrees of freedom:

$$\text{the value of } \tau \cdot n \text{ at two points of each side of } K, \quad (4.3)$$

$$\int_K \tau_{ij} dx, \quad i, j = 1, 2, \quad (4.4)$$

where $\tau \cdot n = (\tau_{11} n_1 + \tau_{12} n_2, \tau_{21} n_1 + \tau_{22} n_2)$ and $n = (n_1, n_2)$ is the outward unit normal to the boundary of K . The space \mathcal{H}_h can now be defined:

$$\mathcal{H}_h = \{ \tau : \tau|_K \in H_K, K \in \mathcal{C}_h, \text{ and } \operatorname{div} \tau \in Y_2 \}.$$

If $\tau|_K \in H_K$, $K \in \mathcal{C}_h$, then $\operatorname{div} \tau \in Y_2$ if and only if $\tau \cdot n$ is continuous at the interelement boundaries, i. e., if for any side S common to the triangles K and K' ,

$$\tau|_K \cdot n = \tau|_{K'} \cdot n \quad \text{on } S,$$

where n is a normal to S . Therefore the degrees of freedom for an element $\tau \in H_h$ can be chosen as follows: the value of $\tau \cdot n$ at two points at each side of \mathcal{C}_h and the values given by (4.4) for $K \in \mathcal{C}_h$.

Finally, defining

$$Y_h = \{v \in Y_2 : v \text{ is linear on } K, K \in \mathcal{C}_h\},$$

the particular case of the finite element method (4.1) we want to consider has been fully described.

In addition to (4.2) the spaces \mathcal{H}_h and Y_h have the following property (see [5]) important for the analysis: There is an interpolation operator $\pi_h : \mathcal{H}_2 \rightarrow \mathcal{H}_h$ such that

$$(\operatorname{div} \pi_h \tau, v) = (\operatorname{div} \tau, v), \quad v \in Y_h, \tag{4.5}$$

and

$$\|\tau - \pi_h \tau\| \leq C h^2 \|\tau\|_{2,2}.$$

Given $\tau \in \mathcal{H}_2$, sufficiently regular e. g. $\tau \in [W_1(\Omega)]^4$, the interpolant $\pi_h \tau$ is defined to be the unique element in H_h satisfying for any side S of \mathcal{C}_h with normal n ,

$$\int_K ((\tau - \pi_h \tau) \cdot n) \cdot v ds = 0 \quad \text{for } v \text{ linear,}$$

and for any $K \in \mathcal{C}_h$,

$$\int_K (\tau_{ij} - (\pi_h \tau)_{ij}) ds = 0, \quad i, j = 1, 2.$$

We observe that if $\operatorname{div} \tau = 0$ in Ω then by (4.2) also $\operatorname{div} \pi_h \tau = 0$ in Ω .

Existence of a solution of the finite element problem can be proved under the following "discrete safe load hypothesis":

$$\left. \begin{aligned} \text{There exists } \delta > 0 \text{ and } \chi_h \in \mathcal{P}_h \cap E \text{ such that} \\ \operatorname{dist}(\chi_h(x), \partial D) \geq \delta, \quad x \in \Omega, \\ \|\chi_h\| \leq C, \end{aligned} \right\} \tag{4.6}$$

where C and δ are independent of h . We note that if the χ in the safe load hypothesis (1.14) for the continuous problem is sufficiently smooth (e. g. if χ is continuous), then (4.6) will be true for h sufficiently small if we choose $\chi_h = \pi_h \chi$.

Let us now consider the convergence of the finite element method (4.1). We have the following result on weak convergence:

THEOREM 3: *If (4.6) holds, then for any p , $1 \leq p < 2$, there exists $(\sigma, u, U) \in \mathcal{P}_q \times Y_p \times R$ satisfying (1.13) with $\mathcal{P} = \mathcal{P}_q$ and $Y = Y_p$, where $(1/q) + (1/p) = 1$, and a sequence $\{h_i\}$ tending to zero, such that*

$$\sigma_h \rightarrow \sigma \text{ weakly in } H \text{ as } h \rightarrow 0,$$

$$u_{h_i} \rightarrow u \text{ weak star in } Y_p,$$

$$U_{h_i} \rightarrow U.$$

Proof: The theorem follows easily from the following *a priori* estimates:

$$\|\sigma_h\| \leq C, \quad (4.7)$$

$$\|u_h\| \leq C, \quad (4.8)$$

$$|U_h| \leq C. \quad (4.9)$$

The estimate (4.7) follows directly by taking $\tau = \chi_h$ in (4.1 a), where χ_h is given by (4.6).

Next, (4.9) follows by choosing $\tau = \chi_h + \pi_h \bar{\chi}$ in (4.1 a), where $\bar{\chi} \in C^\infty(\Omega) \cap H$ satisfies

$$\operatorname{div} \bar{\chi} = 0 \text{ in } \Omega,$$

$$\int_{\Gamma_1} \bar{\chi}_{22} ds \neq 0,$$

$$\|\bar{\chi}\|_\infty \leq \frac{\delta}{2}.$$

Such a $\bar{\chi}$ can easily be constructed by solving a suitable linear elastic problem.

Finally, (4.8) will follow by choosing $\tau = \chi_h + \pi_h \tilde{\chi}$ in (4.1 a), where $\tilde{\chi} \in \mathcal{H}_2$ satisfies

$$\operatorname{div} \tilde{\chi} = g,$$

where g ranges over the ball $\{g \in Y_q : \|g\|_q \leq \mu\}$, with $\mu > 0$ sufficiently small. For instance one can choose $\tilde{\chi} = \varepsilon(\tilde{u})$, where \tilde{u} satisfies

$$\begin{aligned} \operatorname{div}(\varepsilon(\tilde{u})) &= g \text{ in } \bar{\Omega}, \\ \varepsilon(\tilde{u}) \cdot n &= 0 \text{ on } \partial\bar{\Omega}, \end{aligned}$$

and $\bar{\Omega}$ is a domain with smooth boundary such that $\Omega \subset \bar{\Omega}$, $\Gamma_1 \subset \partial\bar{\Omega}$ and g is suitably extended outside Ω . To see that $\tau = \chi_h + \pi_h \tilde{\chi} \in \mathcal{P}_h$ for μ sufficiently

small, we note that by elliptic regularity (see [6]) one has

$$\|\tilde{\chi}\|_{1,q} \leq C \|g\|_q,$$

which by Sobolev's imbedding theorem implies that $\tilde{\chi}$ is continuous and

$$\|\tilde{\chi}\|_{\infty} \leq C \|g\|_q.$$

Now, (4.1 a) and (4.5) together with the previously obtained estimates (4.7) and (4.9) show that

$$(u_h, g) = (u_h, \operatorname{div} \chi) = (u_h, \operatorname{div} \pi_h \tilde{\chi}) \leq C,$$

if $\|g\|_q \leq \mu$, which proves (4.8). This completes the proof.

Finally, we shall obtain an estimate for $\sigma - \sigma_h$ in terms of the quantity

$$\alpha = \inf \{ \beta : \exists \tau_h \in \mathcal{H}_h \cap E \text{ such that } (1 - \beta) \tau_h \in P \text{ and } \|\sigma - \tau_h\| \leq \beta \}.$$

If σ is sufficiently regular then by choosing $\tau_h = \pi_h \sigma$ we see that $\alpha \rightarrow 0$ as $h \rightarrow 0$.

THEOREM 4: *If (4.6) holds then there exists a constant C independent of h such that for $\alpha < 1$,*

$$\|\sigma - \sigma_h\| \leq C \sqrt{\alpha}.$$

Proof: Choosing $\tau = (1 - 2\alpha) \tau_h$ in (4.1 a) where $\tau_h \in \mathcal{H}_h \cap E$ satisfies $\|\sigma - \tau_h\| \leq 2\alpha$, and $\tau = \sigma_h$ in (1.13 a) and adding, we easily find that

$$\frac{1}{C} \|\sigma - \sigma_h\|^2 \leq \|\sigma - \sigma_h\|_a^2 \leq a(\sigma_h, (1 - 2\alpha) \tau_h - \sigma).$$

$$+ 2\alpha U_h F + \int_{\Gamma_1} \varphi(\sigma_{22} - \tau_{22,h}) ds + 2\alpha \int_{\Gamma_1} \varphi \tau_{22,h} ds.$$

Thus, using the estimates (4.7), (4.9) and (1.24) we obtain the desired estimate.

REFERENCES

1. G. DUVAUT, Problèmes de contact entre corps solides déformables, *Applications of Methods of Functional Analysis to Problems in Mechanics*, Joint Symposium IUTAM/IMU, Marseille, 1975, Lecture Notes in Mathematics, Springer Verlag, Berlin-Heidelberg, 1976.
2. G. DUVAUT and L.-J. LIONS, *Les inéquations en Mécanique et en Physique*, Dunod, Paris, 1972.
3. C. JOHNSON, *Existence Theorems for Plasticity Problems*, J. Math. pures et appl., 55, 1976, pp. 431-444.

4. C. JOHNSON, *On Plasticity with Hardening* (to appear in *Mathematical Analysis and Applications*).
5. C. JOHNSON and B. MERCIER, *Some Equilibrium Finite Element Methods for Two-Dimensional Elasticity Problems*, Research report 77.08, Computer Sciences Dept., Chalmers Univ. of Techn., Göteborg.
6. J. NECAS, *Les Méthodes Directes en Théorie des Équations Elliptiques*, Masson, Paris, 1976.